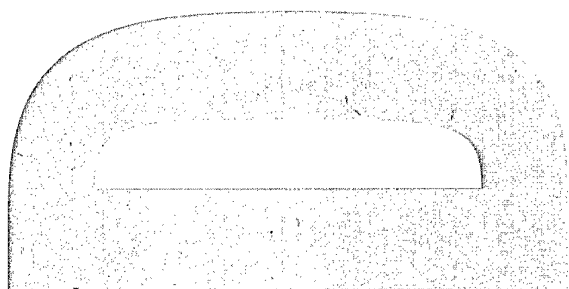
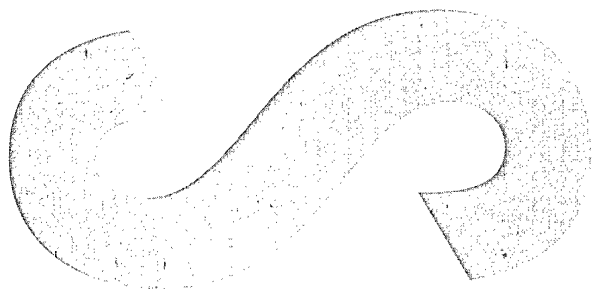


## **Stress Analysis of the F-111 Wing Pivot Fitting**

S. Weller and M. McDonald

DSTO-TN-0271



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# Stress Analysis of the F-111 Wing Pivot Fitting

*S. Weller and M. McDonald*

**Airframes and Engines Division  
Aeronautical and Maritime Research Laboratory**

DSTO-TN-0271

## ABSTRACT

A number of high stress and potential cracking sites in the F-111 wing pivot fitting (WPF) have been identified using a validated 3D finite element model. Selected locations have been analysed in detail, and ranked according to the magnitude of the peak stress. These locations have also been compared with known sites of in-service cracking. Overall, there is very good agreement between the locations identified from the finite element model and those experiencing in-service cracking. The results from this investigation may assist the RAAF in reviewing inspection requirements for the F-111 WPF.

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*Published by*

*DSTO Aeronautical and Maritime Research Laboratory  
PO Box 4331  
Melbourne Victoria 3001 Australia*

*Telephone: (03) 9626 7000  
Fax: (03) 9626 7999  
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# Stress Analysis of the F-111 Wing Pivot Fitting

## Executive Summary

The wing pivot fitting (WPF) of the F-111 aircraft has experienced in-service fatigue cracking at a number of locations. Currently, the Royal Australian Air Force (RAAF) monitor a number of potential cracking sites in the WPF. It is possible that some sites are being monitored too rigorously, causing considerable undue expense for the RAAF. It is also possible that there exist potential cracking sites that are not included for inspection at all.

Recently, AMRL have acquired and developed a validated finite element (FE) model of the F-111 WPF. In this investigation the model has been used to identify potential cracking sites and list them in an approximate order of severity. The identified sites have been compared to in-service cracking experience and known durability and damage tolerance assessment (DADTA) control points.

In-service cracking has occurred in regions of reducing stiffener height known as stiffener runouts (SRO), and around elongated holes known as fuel flow vent holes (FFVH). These features can cause significant local stress concentrations and have been a main focus of this investigation.

The results confirm FFVH 13 and SRO 2U, which are current DADTA control points, as being the most severe locations in the WPF in terms of cracking potential. Other locations identified in the investigation which also have in-service cracking experience, are FFVHs 11, 12, and 14, and SROs 3U and 4U. Some of the locations identified during the investigation have not been the site of in-service cracking. These are FFVHs 15, 16, 57, and 58, SROs 5U and 3L, and the aft outboard edge of the lower plate.

Overall, there is very good agreement between the locations identified in this investigation and locations with in-service cracking experience. The results from this work are considered to be useful for the RAAF for the through-life management of the F-111 WPF.

It should be noted that this investigation only assesses the relative severity of selected locations based on stress level. To gain a more complete picture, a DADTA would be needed to establish the severity in terms of crack growth and inspection interval.

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## 1. Introduction

The F-111 aircraft have experienced in-service cracking at a number of locations in the wing pivot fitting (WPF). Currently, the Royal Australian Air Force (RAAF) monitor a number of potential cracking sites in the WPF. It is possible that some sites are being monitored too rigorously, causing considerable undue expense for the RAAF. It is also possible that there exist potential cracking sites that are not included for inspection at all.

Recently, AMRL have acquired and developed a validated finite element (FE) model of the F-111 WPF [1, 2]. In this investigation the model has been used to identify potential cracking sites and list them in an approximate order of severity. The identified sites have been compared to in-service cracking experience and known durability and damage tolerance assessment (DADTA) control points. The results from this investigation may assist the RAAF in reviewing inspection requirements for the F-111 WPF.

## 2. Background

### 2.1 Wing Pivot Fitting

The location of the WPF is shown in Figure 1. It is designed to transmit wing loads to the wing carry through box, and as such it has high structural importance. The use of D6ac steel in its construction offers high strength but fairly low fracture toughness, making it sensitive to fatigue cracking, especially in areas of high stress concentration. To ensure the safety of the aircraft, in-service wings undergo extensive periodic inspections, and a periodic proof load test known as the cold proof load test (CPLT)<sup>1</sup>. The purpose of the CPLT is to confirm that any undetected cracks are not critical, thus clearing the aircraft for a further period of safe flight.

Application of CPLT loads (approximately design limit loads for the wing) causes localised yielding in some areas of high stress concentration in the WPF, leaving residual stresses in the yielded regions upon unloading. In particular, for positive CPLT loads, certain areas in the upper plate experience compressive yield, with the resultant residual stresses being tensile. It has been found that these residual tensile stresses are the main factor driving in-service fatigue cracks. Although the effect of residual stresses on crack growth rate is significant, it has only been considered qualitatively in this report.

---

<sup>1</sup> In CPLT the aircraft is cooled to -40°F to embrittle the D6ac steel structure, then loaded to -2.4 g and +7.33 g at 56° wing sweep angle followed by -3.0 g and +7.33 g at 26° wing sweep angle.

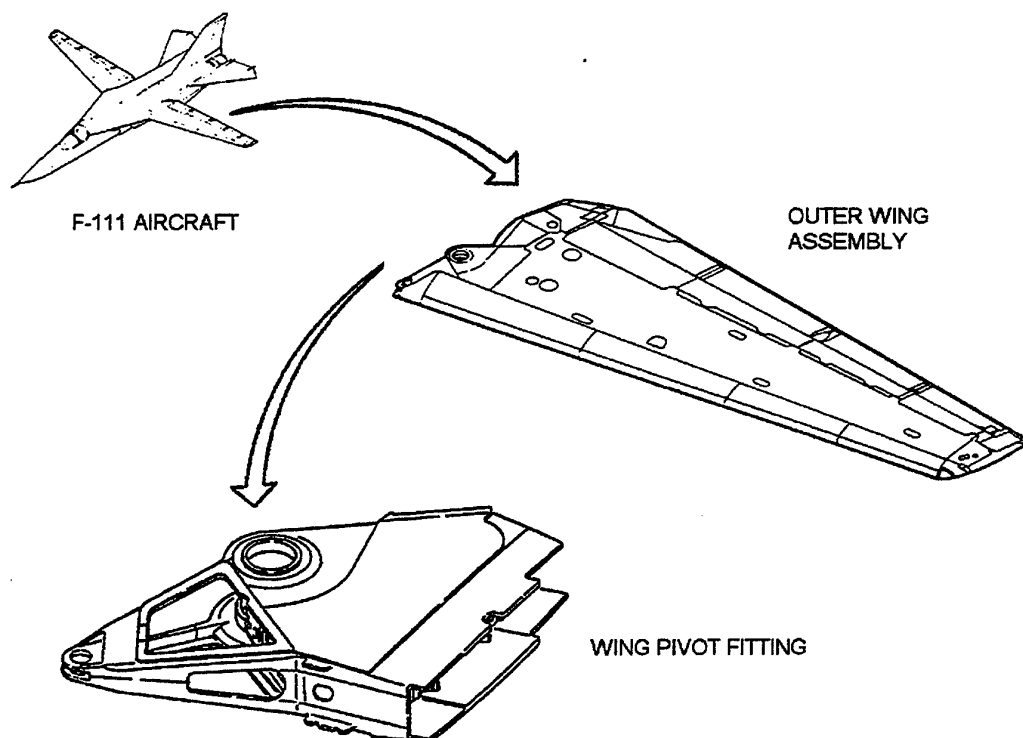


Figure 1: Location of the F-111 wing pivot fitting

Cracking has occurred in regions of reducing stiffener height known as stiffener runouts (SRO), and around elongated holes known as fuel flow vent holes (FFVH). Areas of local stress concentration, referred to here as 'hot spots', are predominantly found at FFVHs and SROs, hence these features have been a main focus of this investigation. Figure 2 shows the locations of all FFVHs and SROs in the WPF.

Two well-known critical sites in the upper plate of the WPF are FFVH 13 and SRO 2U<sup>2</sup>. Both exhibit very high stress concentrations and both have experienced numerous in-service cracks [3, 4, and 5]. Furthermore, these items are two of the control points for the F-111 DADTA. A significant amount of work has been directed toward managing these sites [4-11]. Recent instances of cracking at FFVHs 11 and 12 [12] indicate that other areas may also be cause for concern, especially as the aircraft moves into the latter stages of its planned operational life.

<sup>2</sup> The labelling of FFVHs and SROs used in this report follows that used in the aircraft structural repair manual, where 'U' and 'L' are used to differentiate between upper and lower plate stiffener runouts. SRO 2U is often referred to as simply SRO 2, or SRO#2.

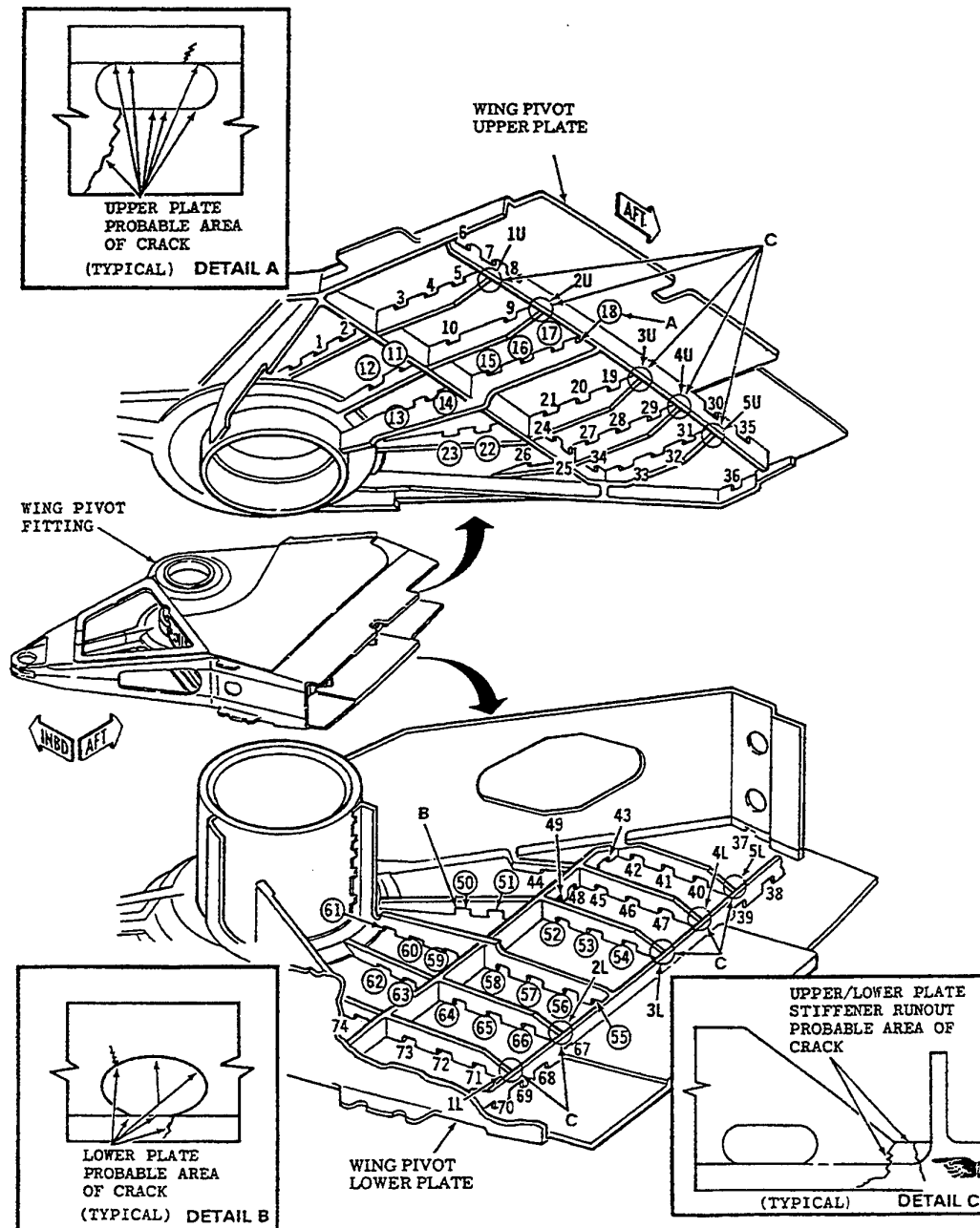


Figure 2: F-111 wing pivot fitting details



## 2.2 Finite Element Model

The original F-111 WPF FE model was supplied by Lockheed Martin Tactical Aircraft Systems (LMTAS) [1]. A section of the model is shown in Figure 3. It was designed to capture the internal load distribution within the WPF structure. The WPF structure is modelled in detail with virtually all WPF components represented using 3D solid elements. A small stub of wing is also coarsely modelled to help apply the wing loads to the WPF correctly. The model is configured with a lower plate boron doubler.

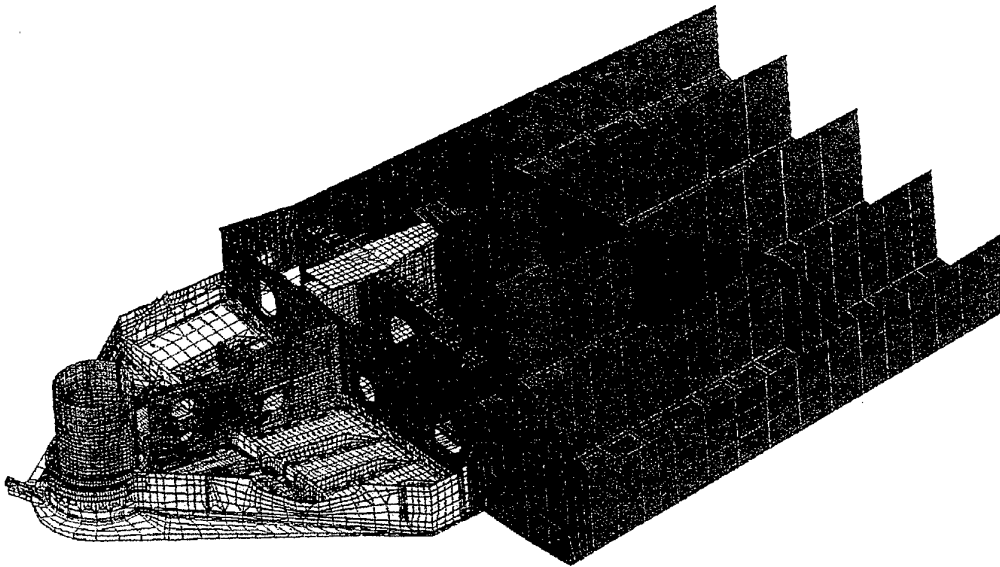


Figure 3: *F-111 WPF finite element model*

The mesh refinement is suitable for capturing the stiffness of the structure. It may also be used to obtain stresses and strains at far field locations not subject to high stress gradients. Due to the general coarseness of the mesh, it cannot capture accurate stress or strain results at local notch details such those found in the FFVHs and SROs.

This model has been validated with experimental results from an ex-USAF test wing [13]. As supplied, the model correlated well at far field locations such as the upper and lower plate. However, the strains near to FFVH 13 and the SROs correlated poorly. Consequently, AMRL has since made various enhancements to the model, and improved the correlation at FFVH 13 and SRO 2U significantly [2]. The current improved model was used as the base model for the study reported here.

Far field upper and lower plate strains from the current WPF FE model agree in general to within 10% of test results. The correlation is best toward the middle of the WPF (0% - 10% difference), and tends to degenerate at the fore and aft extremities (up to 30% difference). The FE strains on the integral WPF stiffeners also agree well with test results, the average difference being less than 10%. This includes stiffener #3, which contains FFVH 13. The FE strains near SRO 2U are also within 10% of test results. A similar correlation is expected at other SROs, however it is difficult to verify conclusively using the limited test results that are available.

The model is expected to be very useful for identifying hot spots in the structure. However, given that the level of correlation does vary somewhat across the model, direct comparison of stresses at each location should be interpreted with some discretion.

The FFVH and SRO profiles used for the investigation are shown in Appendix A. These are the profiles given by the assembly drawings for the WPF upper and lower plates, and they are labelled 'blueprint' to distinguish them from the range of profiles that have been implemented in the model for separate investigations.

### 3. Analysis Approach

The investigation was conducted in two parts. Firstly, the current FE model was used as is to identify potential cracking sites. All stress results for this part are termed 'preliminary'. Secondly, mesh improvements were made at selected hot spots to determine the peak stress so that they could be ranked in order of severity. All stress results for the second part are termed 'final'.

The load case used to compare peak stresses was the +7.33 g CPLT wing tip up condition. In general, this is the most severe loading for the WPF, and at isolated locations localised yielding can occur. The FE analyses reported here are linear elastic only - yielding is not modelled. However, the linear elastic results are still adequate for comparing the relative stress severity of hot spots.

### 4. Identification of Potential Cracking Sites

Prior to any mesh refinements, stress results were extracted from the current model to identify areas requiring further investigation. The peak von Mises stresses from each FFVH and SRO were tabulated. This, in combination with a qualitative visual

inspection of stress contour plots (Figures 4 and 5) and an examination of full-scale test data, was used to select the most severe locations.

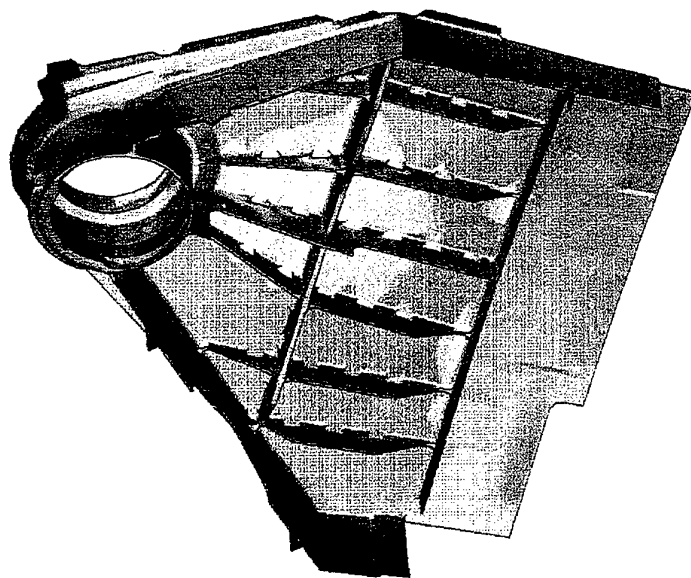


Figure 4: *Stress contour plot of the WPF upper plate*

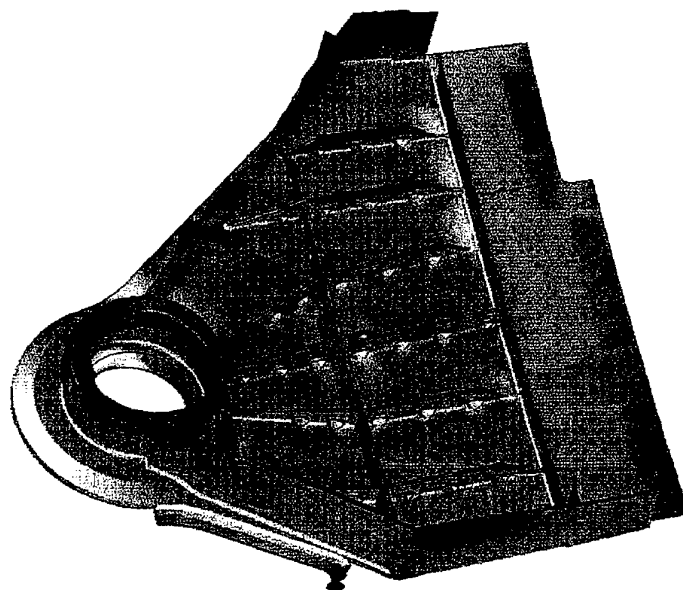


Figure 5: *Stress contour plot of the WPF lower plate*

The peak stresses occurred at locations along the FFVHs and SROs where in-service cracking also occurs, as shown in Figure 2. These locations are the lower inboard corner of the upper plate FFVHs, the upper inboard corner of the lower plate FFVHs, and the corner radius of the SROs, both in the upper and lower plates. The upper plate peak stresses are compressive while the lower plate peak stresses are tensile.

It was noted that the quality and density of the FE mesh was not consistent across the selected locations, and as expected, the majority exhibited coarse meshes incapable of capturing the detail stress field. Nonetheless, qualitative inspection of the stress results was adequate to identify the hot spots.

Initially, all locations exhibiting stress levels above 80% of the relevant DADTA control point level<sup>3</sup> were included in the selected set. Since very few FFVHs met this criterion based on the preliminary results, the investigation was broadened to encompass a group of the next most severe vent holes.

Figures 6 and 7 show the preliminary peak stresses at each FFVH and SRO. The results are also tabulated in Appendix B. On the basis of these results the following items were selected for further investigation:

- FFVHs 11, 12, 13, 14, 15, and 16 in the WPF upper plate
- FFVHs 57 and 58 in the WPF lower plate
- SROs 2, 3, 4 in the WPF upper plate
- SRO 3 in the WPF lower plate

The location of these items in the WPF can be seen in Figure 2.

SRO 5 in the upper plate was also included in this group after a review of full-scale strain survey data [13]. For the +7.33 g load case the test data clearly shows yield conditions for this SRO similar to SRO 3 and SRO 4. This was not evident from the preliminary FE results.

The magnitude of the peak stress at SRO 3L is approximately 80% of the SRO 2U value and the stresses there are tensile for positive loads. Although it may experience a relatively high tensile spectrum during flight conditions, it is not considered as critical due to an expected residual compressive field after CPLT loading. This would have a beneficial effect on fatigue crack growth. A similar scenario would apply to all lower plate SROs. For this reason, no detail stress analysis has been performed at SRO 3L in this investigation, as it may be inappropriate to use the stress results alone to compare this location with upper plate SROs. A DADTA analysis, which would account for the residual compressive stress field, would be required in order to assess correctly its relative cracking potential.

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<sup>3</sup> FFVH 13 for upper vent holes, FFVH 58 for lower vent holes, SRO 2U for runouts.

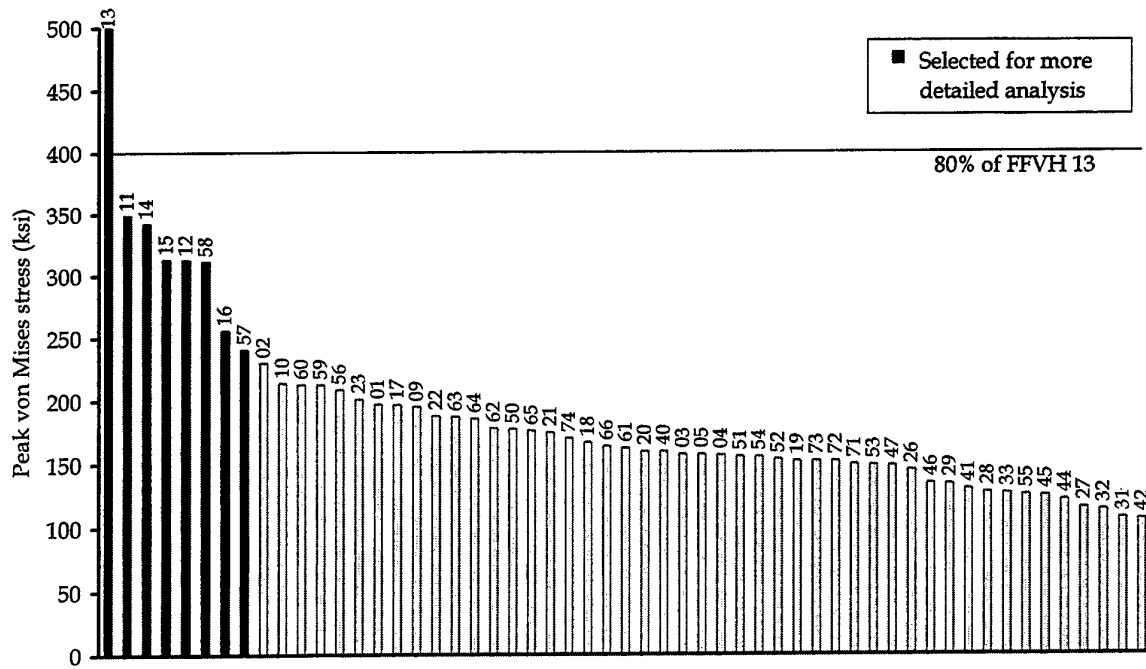


Figure 6: Preliminary peak stress results for each fuel flow vent hole (+7.33 g load case)

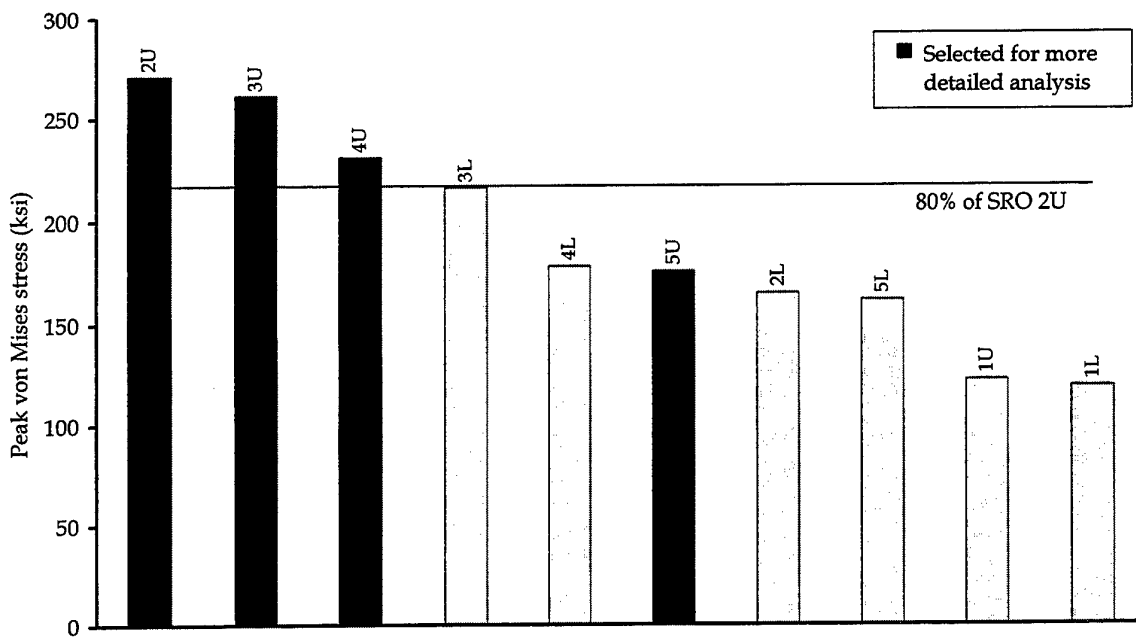


Figure 7: Preliminary peak stress results for each stiffener runout (+7.33 g load case)

One other area of relatively high stress indicated by the preliminary results is the aft edge of the lower plate immediately inboard of bulkhead 1.0 (Figure 8). The thickness of the lower plate increases from 0.185" inboard of the bulkhead to 0.295" where the plate joins the lower wing skin. The peak stress occurs in the fillet radius of this thickness change and is tensile. The von Mises stress of 238.8 ksi is comparable with the preliminary results for SRO 4U.

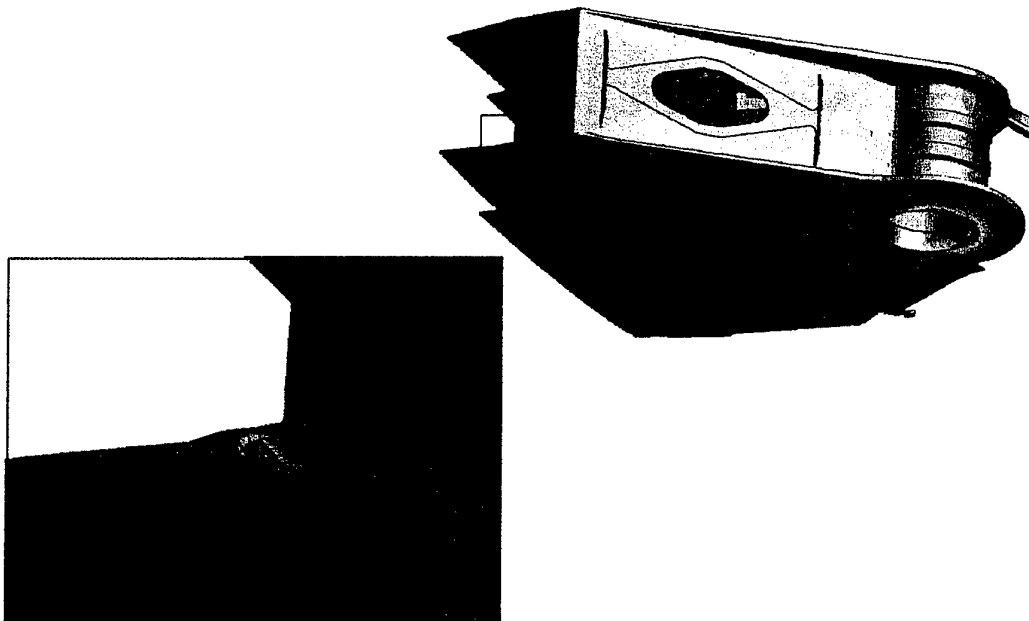


Figure 8: *Localised stress peak in the aft edge of the lower plate*

The cause of this stress peak has not been fully determined. An inspection of the FE model in this region reveals a complex meeting of parts, involving the WPF lower plate, the lower wing skin, bulkhead 1.0 and spars. This suggests that the stress peak could be a product of the way these parts and their connections have been approximated in the model. Alternatively it may be a real effect. It was noted that a comparison of far field lower plate strains in the vicinity indicates that the FE results overestimate full-scale test results by around 30%. Subsequently, this location was not investigated any further.

## 5. Detail Stress Analysis of Potential Cracking Sites

As mentioned previously, extensive mesh refinement was required at most of the selected hot spots to capture accurate peak stresses. In general, this involved replacing irregular or distorted elements and increasing the mesh density. Higher order elements, and convergence checks, were also applied to the areas of interest to ensure that mesh discretisation errors were minimised.

Figures 9 and 10 show typical stress contour plots for FFVH 13 and SRO 2U respectively.

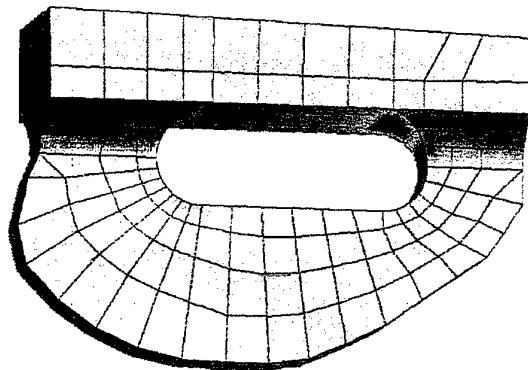


Figure 9: Stress contour plot of FFVH 13

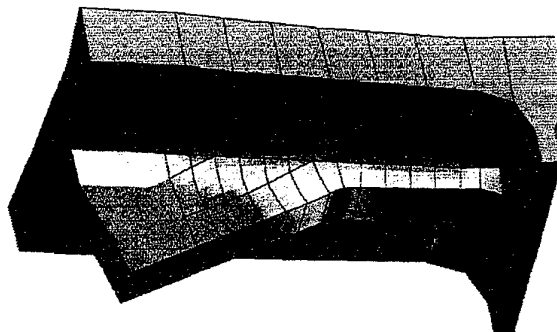


Figure 10: Stress contour plot of SRO 2U

## 5.1 Comparison of Selected Fuel Flow Vent Holes

Table 1 summarises the final peak von Mises stress at each FFVH as determined from the detail stress analysis. These are represented graphically in Figure 11, with the preliminary results included for comparison. As can be seen, the mesh improvements have resulted in a marked increase in the peak stress. For FFVHs 13 and 58 the increase is relatively small since the initial meshes at these locations were of reasonable quality. Conversely, there is a large increase in the peak stresses indicated at FFVHs 15 and 16, for which the initial meshes were very poor.

Table 1: *Final peak stresses at selected fuel flow vent holes*

| FFVH | Peak von Mises Stress (ksi) | % of FFVH 13 | In-service Cracking | DADTA Item |
|------|-----------------------------|--------------|---------------------|------------|
| 13   | 553.2                       | 100          | ✓                   | ✓          |
| 11   | 445.6                       | 81           | ✓                   |            |
| 15   | 442.2                       | 80           |                     |            |
| 14   | 427.4                       | 77           | ✓                   |            |
| 12   | 410.4                       | 74           | ✓                   |            |
| 16   | 371.3                       | 67           |                     |            |
| 58   | 339.2                       | 61           |                     | ✓          |
| 57   | 318.7                       | 58           |                     |            |

As expected FFVH 13 shows the most severe peak stress. FFVHs 11, 12, 14, and 15 exhibit approximately similar stress levels at about 80% of the FFVH 13 level. FFVH 16 is less severe at 67% while FFVHs 57 and 58 are around 60%.

For each upper plate FFVH (11 to 16) the peak stress occurs in the lower inboard corner, and is compressive for the load case considered. A significant but lesser peak (typically 80% of lower inboard value) is also evident in the upper outboard corner. For each lower plate FFVHs (57 and 58) the peak stress occurs in the upper inboard corner, and is tensile for the load case considered.

Table 1 also highlights vent holes that are known to have experienced in-service cracking in the RAAF fleet. Numerous cracks have been found at FFVH 13, while only a few cracks have been found at the other vent holes [3].

Direct comparison of peak elastic stress results between the upper and lower plate FFVHs may be inappropriate. This is because upper plate FFVH peaks are usually compressive, while lower plate FFVH peaks are usually tensile. This not only affects the sign of the normal operating stress, but also the residual stress induced after CPLT loading, which is known to have a considerable effect on crack growth rates. To fully



substantiate the relative cracking potential of all locations in the WPF a DADTA analysis would be required.

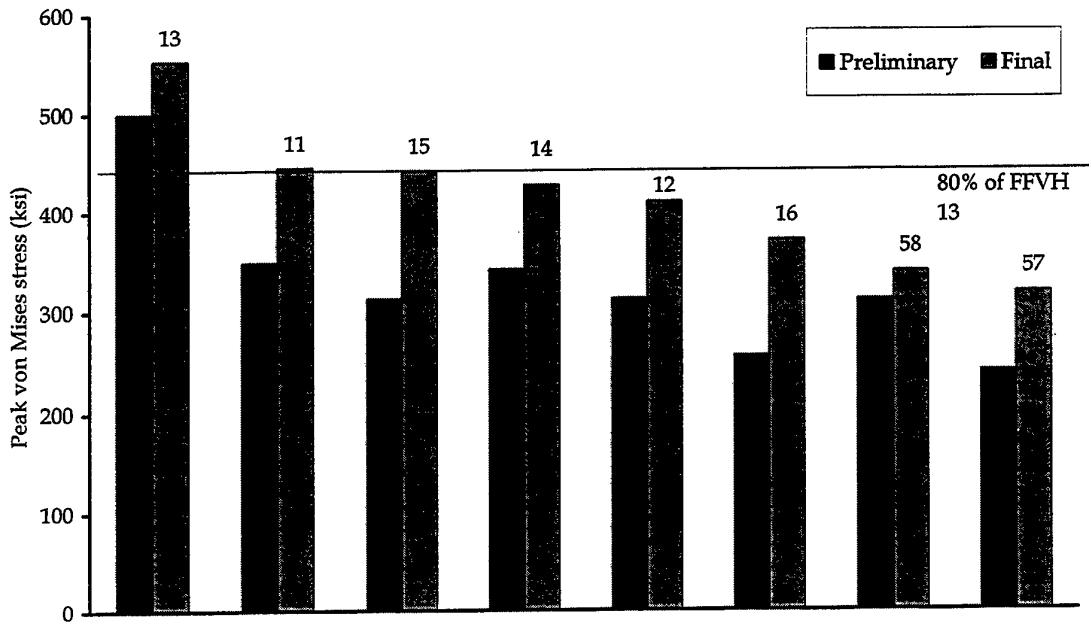


Figure 11: Preliminary and final peak stresses at selected FFVHs (+7.33 g load case)

## 5.2 Comparison of Selected Stiffener Runouts

Table 2 summarises the final peak von Mises stress at each SRO as determined from the detail stress analysis. The results are shown graphically in Figure 12, with the preliminary results included for comparison. As can be seen, SRO 2U is the most severe, followed by SRO 3U, SRO 4U and SRO 5U. The peak stresses at these SROs are compressive for the load case considered.

Table 2: Final peak stresses at selected stiffener runouts

| SRO | Peak von Mises Stress (ksi) | % of SRO 2U | In-service Cracking | DADTA Item |
|-----|-----------------------------|-------------|---------------------|------------|
| 2U  | 484.5                       | 100%        | ✓                   | ✓          |
| 3U  | 417.7                       | 86%         | ✓                   |            |
| 4U  | 356.7                       | 74%         | ✓                   |            |
| 5U  | 342.2                       | 71%         |                     |            |

The detail peak stress results are much larger than the corresponding preliminary results with increases of up to 95%. This is due to the combined effect of the modelling adjustments in the SRO region and the very poor quality of the initial mesh.

Table 2 also highlights the runouts that are known to have experienced in-service cracking. Numerous cracks have been found at SRO 2U, while only a few cracks have occurred at the other runouts [3].

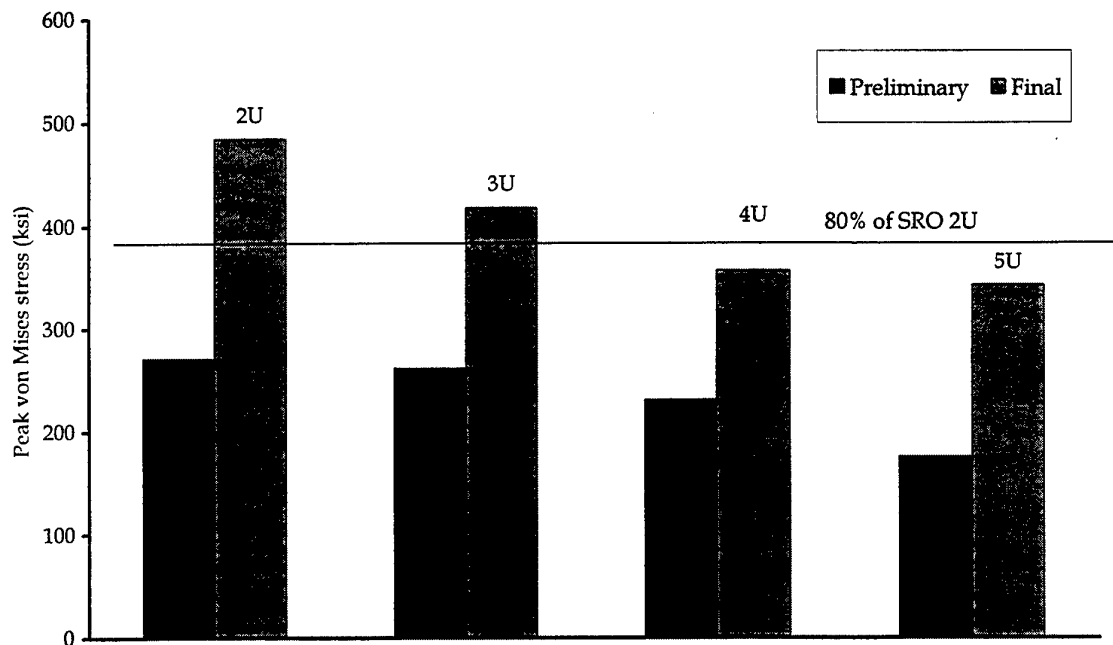


Figure 12: Preliminary and final peak stresses at selected SROs (+7.33 g load case)

## 6. Conclusion

This investigation presents a relative comparison of peak elastic stresses at potential cracking sites in the F-111 wing pivot fitting. The stress results were derived from the current validated FE model of the WPF.

The results confirm FFVH 13 and SRO 2U, which are current DADTA control points, as being the most severe locations in the WPF in terms of cracking potential. Other locations identified in the investigation which also have in-service cracking experience, are FFVHs 11, 12, and 14, and SROs 3U and 4U.

Some of the locations identified during the investigation have not been the site of in-service cracking. These are FFVHs 15, 16, 57, and 58, SROs 5U and 3L, and the aft outboard edge of the lower plate.

Overall, there is very good agreement between the locations identified in this investigation and locations with in-service cracking experience. The results from this work are considered to be useful for the RAAF for the through-life management of the F-111 WPF.

It should be noted that this investigation only assesses the relative severity of selected locations based on stress level. To gain a more complete picture a DADTA would be needed to establish the severity in terms of crack growth rate and inspection interval.

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# Appendix A: Blueprint Profiles

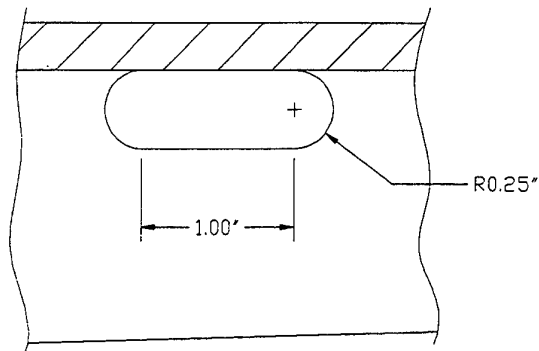


Figure 13: *Blueprint profile for upper plate fuel flow vent holes*

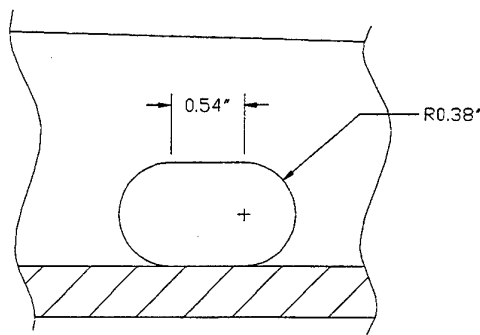


Figure 14: *Blueprint profile for lower plate fuel flow vent holes*

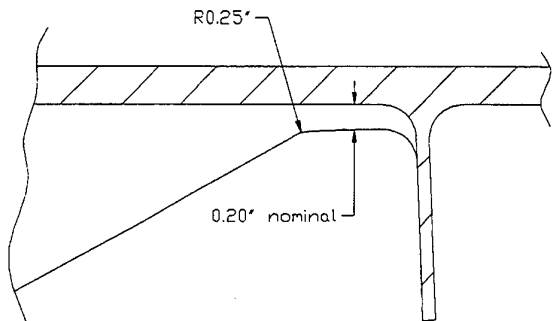


Figure 15: *Blueprint profile for upper plate stiffener runouts*

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## Appendix B: Preliminary peak stress results

This Appendix gives the preliminary peak von Mises stresses at each FFVH and SRO for the +7.33 g CPLT load case. Results were obtained from the Patran result file *wpf\_003\_2d\_h0\_results.db*.

Table A1: Preliminary peak stress results at each FFVH

| FFVH | Peak $\sigma_{VM}$<br>[ksi] | % of FFVH 13 | Stress State | Node ID |
|------|-----------------------------|--------------|--------------|---------|
| 13   | 500.2                       | 100%         | compressive  | 76979   |
| 11   | 349.0                       | 70%          | compressive  | 57171   |
| 14   | 342.3                       | 68%          | compressive  | 58040   |
| 15   | 313.0                       | 63%          | compressive  | 24066   |
| 12   | 312.6                       | 62%          | compressive  | 57287   |
| 58   | 311.4                       | 62%          | tensile      | 5808    |
| 16   | 255.9                       | 51%          | compressive  | 23834   |
| 57   | 240.7                       | 48%          | tensile      | 6584    |
| 02   | 230.1                       | 46%          | compressive  | 56847   |
| 10   | 213.8                       | 43%          | compressive  | 58591   |
| 60   | 212.5                       | 42%          | tensile      | 3908    |
| 59   | 212.4                       | 42%          | tensile      | 3780    |
| 56   | 208.4                       | 42%          | tensile      | 6647    |
| 23   | 201.0                       | 40%          | compressive  | 57572   |
| 01   | 196.9                       | 39%          | compressive  | 57008   |
| 17   | 196.3                       | 39%          | compressive  | 23718   |
| 09   | 194.8                       | 39%          | compressive  | 58655   |
| 22   | 187.7                       | 38%          | compressive  | 57428   |
| 63   | 186.8                       | 37%          | tensile      | 1243    |
| 64   | 185.4                       | 37%          | tensile      | 5250    |
| 62   | 177.7                       | 36%          | tensile      | 1219    |
| 50   | 176.9                       | 35%          | tensile      | 7491    |
| 65   | 175.7                       | 35%          | tensile      | 5284    |
| 21   | 174.4                       | 35%          | compressive  | 24470   |
| 74   | 169.8                       | 34%          | tensile      | 1256    |
| 18   | 166.5                       | 33%          | compressive  | 24009   |
| 66   | 163.4                       | 33%          | tensile      | 5382    |
| 61   | 161.9                       | 32%          | tensile      | 34222   |
| 20   | 159.2                       | 32%          | compressive  | 24339   |
| 40   | 159.1                       | 32%          | tensile      | 15428   |
| 03   | 157.1                       | 31%          | compressive  | 23339   |
| 05   | 157.1                       | 31%          | compressive  | 23455   |
| 04   | 156.3                       | 31%          | compressive  | 23571   |
| 51   | 155.2                       | 31%          | tensile      | 7392    |



Table A1: Preliminary peak stress results at each FFVH, continued

| FFVH | Peak $\sigma_{VM}$<br>[ksi] | % of FFVH 13 | Stress State | Node ID |
|------|-----------------------------|--------------|--------------|---------|
| 54   | 155.0                       | 31%          | tensile      | 11670   |
| 52   | 153.0                       | 31%          | tensile      | 11208   |
| 19   | 151.8                       | 30%          | compressive  | 24569   |
| 73   | 151.8                       | 30%          | tensile      | 8312    |
| 72   | 151.6                       | 30%          | tensile      | 8555    |
| 71   | 149.1                       | 30%          | tensile      | 8798    |
| 53   | 148.4                       | 30%          | tensile      | 11431   |
| 47   | 148.2                       | 30%          | tensile      | 12691   |
| 26   | 144.6                       | 29%          | compressive  | 57768   |
| 46   | 134.2                       | 27%          | tensile      | 12547   |
| 29   | 133.9                       | 27%          | compressive  | 25041   |
| 41   | 129.8                       | 26%          | tensile      | 15433   |
| 28   | 127.1                       | 25%          | compressive  | 24861   |
| 33   | 126.5                       | 25%          | compressive  | 25553   |
| 55   | 125.2                       | 25%          | tensile      | 6397    |
| 45   | 124.7                       | 25%          | tensile      | 12421   |
| 44   | 121.2                       | 24%          | tensile      | 2603    |
| 27   | 115.2                       | 23%          | compressive  | 24975   |
| 32   | 113.6                       | 23%          | compressive  | 25321   |
| 31   | 107.0                       | 21%          | compressive  | 25437   |
| 42   | 106.0                       | 21%          | tensile      | 15439   |

Table A2: Preliminary peak stress results at each SRO

| SRO | Peak $\sigma_{VM}$<br>[ksi] | % of SRO 2U | Stress State | Node ID |
|-----|-----------------------------|-------------|--------------|---------|
| 2U  | 270.8                       | 100%        | compressive  | 158028  |
| 3U  | 261.0                       | 96%         | compressive  | 25951   |
| 4U  | 230.5                       | 85%         | compressive  | 25889   |
| 3L  | 215.8                       | 80%         | tensile      | 12013   |
| 4L  | 178.1                       | 66%         | tensile      | 13355   |
| 5U  | 175.5                       | 65%         | compressive  | 25791   |
| 2L  | 164.7                       | 61%         | tensile      | 5356    |
| 5L  | 161.5                       | 60%         | tensile      | 14521   |
| 1U  | 122.2                       | 45%         | compressive  | 23689   |
| 1L  | 119.1                       | 44%         | tensile      | 8839    |

# Stress Analysis of the F-111 Wing Pivot Fitting

S. Weller and M. McDonald

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| 2. TITLE<br><br>Stress Analysis of the F-111 Wing Pivot Fitting   |  |                             | 3. SECURITY CLASSIFICATION (FOR UNCLASSIFIED REPORTS THAT ARE LIMITED RELEASE USE (L) NEXT TO DOCUMENT CLASSIFICATION)<br><br>Document (U)<br>Title (U)<br>Abstract (U) |  |  |
| 4. AUTHOR(S)<br><br>S. Weller and M. McDonald   |  |                             | 5. CORPORATE AUTHOR<br><br>Aeronautical and Maritime Research Laboratory<br>PO Box 4331<br>Melbourne Vic 3001 Australia   |  |  |
| 6a. DSTO NUMBER<br>DSTO-TN-0271   |  | 6b. AR NUMBER<br>AR-011-422 |   | 6c. TYPE OF REPORT<br>Technical Note                               |  |
| 7. DOCUMENT DATE<br>April 2000  |  |                             |   |  |  |
| 8. FILE NUMBER<br>M1/9/711  |  | 9. TASK NUMBER<br>96/102    |   | 10. TASK SPONSOR<br>DGTA   |  |
| 11. NO. OF PAGES<br>20  |  | 12. NO. OF REFERENCES<br>13 |   |  |  |
| 13. URL on World Wide Web<br><a href="http://www.dsto.defence.gov.au/corporate/reports/DSTO-TN-0271.pdf">http://www.dsto.defence.gov.au/corporate/reports/DSTO-TN-0271.pdf</a>  |  |                             |   | 14. RELEASE AUTHORITY<br><br>Chief, Airframes and Engines Division |  |
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| 18. DEFTEST DESCRIPTORS<br><br>F-111 aircraft, Wing pivot fittings, Fatigue life, Cracking (fracturing), Stress analysis, Finite element analysis, Holes (openings)   |  |                             |   |  |  |
| 19. ABSTRACT<br>A number of high stress and potential cracking sites in the F-111 wing pivot fitting (WPF) have been identified using a validated 3D finite element model. Selected locations have been analysed in detail, and ranked according to the magnitude of the peak stress. These locations have also been compared with known sites of in-service cracking. Overall, there is very good agreement between the locations identified from the finite element model and those experiencing in-service cracking. The results from this investigation may assist the RAAF in reviewing inspection requirements for the F-111 WPF. |  |                             |   |  |  |